## A Robust Solution of the Spherical Burmester Problem

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## Outline

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(2) Synthesis for spherical rigid-body guidance


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(3) A linkage with a P-joint
4. Implementation considerations


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## The Spherical Burmester Problem

What is the Burmester problem?

- The classical Burmester problem

Are there any points in a rigid body whose corresponding position lies on a circle of the fixed plane for the four arbitrarily prescribed positions?-Burmester, 1888

- The spherical Burmester problem

An extension of the classical Burmester problem, in which we are interested in synthesizing a spherical four-bar linkage to visit a discrete set of orientations of the coupler link

## Revelent Works

The nature of the problem is to solve a system of trigonometric equations. The challenges can be seen in

- The numerics
- Branch-defect detection
- Consideration of joint type, R- or P- joint

The problem has been studied by different approaches:

- Geometric: using the Burmester-Roth Theorem to construct center axes-Roth, 1967
- Polynomial approach: sixth degree polynomial can be found through variable elimination - Ruth and McCarthy, 1996


## The proposed approach in this work

Proposed here is a semigraphical approach, consisting of three steps:
(1) Algebraic formulation-problem definition
(2) Graphic display+inspection-raw solution estimates
(3) Numerical solution-accurate result

It eliminates spurious solutions and filters complex solutions
An extension of a previous work on the classical Burmester problem - Chen, Bai and Angeles 2008

## Problem Formulation



Find a spherical four-bar linkage that will conduct its coupler link through a set $\mathcal{S}$ of $m$ attitudes given by the orthogonal matrices $\left\{\mathbf{Q}_{j}\right\}_{1}^{m}$, defined with respect to a reference attitude given by $\mathbf{Q}_{0}=\mathbf{1}$, the $3 \times 3$ identity matrix.

## Synthesis Equations

Assuming all links are rigid, all angles $\alpha_{i}$ remain constant

$\mathbf{a}_{j}^{T} \mathbf{b}=\mathbf{a}_{0}^{T} \mathbf{b} \quad$ or $\quad\left(\mathbf{a}_{j}-\mathbf{a}_{0}\right)^{T} \mathbf{b}=0 ; j=1, \ldots, m$
where

$$
\mathbf{a}_{j}=\mathbf{Q}_{j} \mathbf{a}_{0}
$$

In a compact form, (1) becomes

$$
\begin{equation*}
\mathbf{c}_{j}^{T} \mathbf{b}=0, \quad j=1, \ldots, m \tag{3}
\end{equation*}
$$

with

$$
\begin{equation*}
\mathbf{c}_{j} \equiv\left(\mathbf{Q}_{j}-\mathbf{1}\right) \mathbf{a}_{0} \tag{4}
\end{equation*}
$$

## Synthesis Equation (cont'd)

In invariant form,

$$
\begin{equation*}
\mathbf{Q}_{j}=\mathbf{1}+s_{j} \mathbf{E}_{j}+\left(1-c_{j}\right) \mathbf{E}_{j}^{2}, \quad c_{j} \equiv \cos \phi_{j}, \quad s_{j} \equiv \sin \phi_{j} \tag{5}
\end{equation*}
$$

where $\mathbf{E}_{j}$ denotes the cross-product matrix (CPM) of $\mathbf{e}_{j}$, the unit vector that defines the direction of the axis of rotation of $\mathbf{Q}_{j}$, and $\phi_{j}$ the angle of rotation. The synthesis equation becomes

$$
\begin{equation*}
\mathbf{a}_{0}^{T} \mathbf{E}_{j}\left[s_{j} \mathbf{1}-\left(1-c_{j}\right) \mathbf{E}_{j}\right] \mathbf{b}=0, \quad j=1, \ldots, m \tag{6}
\end{equation*}
$$

whose solution depends on the number $m$ of prescribed poses.

## Synthesis with three poses

In this case, $m=2$, i.e., two constraint equations occur:

$$
\begin{equation*}
\mathbf{c}_{1}^{T} \mathbf{b}=0, \quad \text { and } \quad \mathbf{c}_{2}^{T} \mathbf{b}=0 \tag{7}
\end{equation*}
$$

One of the two vectors $\mathbf{a}_{0}$ and $\mathbf{b}$ can be prescribed arbitrarily. In the case of $\mathbf{a}_{0}$ known, then

$$
\begin{equation*}
\mathbf{b}=\frac{\mathbf{c}_{1} \times \mathbf{c}_{2}}{\left\|\mathbf{c}_{1} \times \mathbf{c}_{2}\right\|} \tag{8}
\end{equation*}
$$

A similar reasoning, with obvious modifications, follows if $\mathbf{b}$ is prescribed.

## Synthesis with four poses

Now we have $m=3$, the constraints being

$$
\begin{equation*}
\mathbf{c}_{1}^{T} \mathbf{b}=0, \quad \mathbf{c}_{2}^{T} \mathbf{b}=0 \quad \text { and } \quad \mathbf{c}_{3}^{T} \mathbf{b}=0 \tag{9}
\end{equation*}
$$

All three vectors $\mathbf{c}_{j}$ must be coplanar, and hence

$$
\begin{equation*}
F\left(\mathbf{a}_{0}\right) \equiv \mathbf{c}_{1} \times \mathbf{c}_{2} \cdot \mathbf{c}_{3}=0 \tag{10}
\end{equation*}
$$

This is a cubic, homogeneous equation in $\mathbf{a}_{0}$, which defines a conical cubic surface whose apex is the origin.

(a)

(b)

Figure: (a) the circlepoint curve and (b) the centerpoint curve

## Synthesis with five poses

Now $m=4$, the synthesis equations can be cast into the form

$$
\underbrace{\left[\begin{array}{c}
\mathbf{a}_{0}^{T} \mathbf{E}_{1}\left[s_{1} \mathbf{1}-\left(1-c_{1}\right) \mathbf{E}_{1}\right]  \tag{11}\\
\mathbf{a}_{0}^{T} \mathbf{E}_{2}\left[s_{2} \mathbf{1}-\left(1-c_{2}\right) \mathbf{E}_{2}\right] \\
\mathbf{a}_{0}^{T} \mathbf{E}_{3}\left[s_{3} \mathbf{1}-\left(1-c_{3}\right) \mathbf{E}_{3}\right] \\
\mathbf{a}_{0}^{T} \mathbf{E}_{4}\left[s_{4} \mathbf{1}-\left(1-c_{4}\right) \mathbf{E}_{4}\right]
\end{array}\right]}_{\equiv \mathbf{C}} \mathbf{b}=\mathbf{0}_{4}
$$

Non-trivial solutions require a rank-deficient C:

$$
\begin{equation*}
\Delta_{j}\left(\mathbf{a}_{0}\right) \equiv \operatorname{det}\left(\mathbf{C}_{j}\right)=0, \quad j=1, \ldots, 4 \tag{12}
\end{equation*}
$$

$\mathbf{C}_{j}$ being the $3 \times 3$ matrix obtained from $\mathbf{C}$ upon deleting its $j$ th row. Equations (12) define four conical cubic surfaces with one common apex.

## Synthesis with one P-joint

A spherical linkage may end up with a P-joint, i.e., a slider moving on a circular guide, similar to the planar crank-slider mechanism.
We consider this a special case, with $\alpha_{4}=90 \mathrm{deg}$. Thus

$$
\begin{equation*}
\mathbf{b}^{T} \mathbf{Q}_{j} \mathbf{a}_{0}=0, \quad j=0, \ldots, m \tag{13}
\end{equation*}
$$

The above $m$ equations can be written as

$$
\begin{equation*}
\mathbf{H a}_{0}=\mathbf{0}_{n} \tag{14}
\end{equation*}
$$

where $\mathbf{H}$ is a $n \times 3$ matrix with $n=m+1$. Non-trivial solutions for $\mathbf{b}$ require a rank-deficient $\mathbf{H}$, i.e., for $m=3$,

$$
\begin{equation*}
\operatorname{det}\left(\mathbf{H}_{i}\right)=0, \quad i=1, \ldots, 4 \tag{15}
\end{equation*}
$$

where the $3 \times 3$ matrix $\mathbf{H}_{i}$ is obtained by removing the $i$ th row from matrix $\mathbf{H}$.

## Representation of the axes of rotation

We shall use spherical coordinates on the unit sphere, namely, longitude and latitude.


Let $\vartheta_{A}$ and $\varphi_{A}$ be the longitude and the latitude of $A_{0}, \vartheta_{B}$ and $\varphi_{B}$ those of $B$. Hence,

$$
\mathbf{a}_{0}=\left[\begin{array}{c}
\cos \varphi_{A} \cos \vartheta_{A} \\
\cos \varphi_{A} \sin \vartheta_{A} \\
\sin \varphi_{A}
\end{array}\right], \quad \mathbf{b}=\left[\begin{array}{c}
\cos \varphi_{B} \cos \vartheta_{B} \\
\cos \varphi_{B} \sin \vartheta_{B} \\
\sin \varphi_{B}
\end{array}\right]
$$

The ranges of all spherical coordinates:
$\left\{\varphi_{A}, \vartheta_{A}, \varphi_{B}, \vartheta_{B}\right\} \in[-\pi / 2, \pi / 2]$.

## Branching detection

Branching-defect can occur in spherical linkages.
The branching defect is detected here by means of the sign of the sine of the transmission angle, in analogy with the planar case:

$$
\begin{equation*}
\sin \mu=\|\overline{\mathbf{a}} \times \overline{\mathbf{b}}\| /(\|\overline{\mathbf{a}}\|\|\overline{\mathbf{b}}\|) \tag{17}
\end{equation*}
$$

where $\overline{\mathbf{a}}=\mathbf{a}-\left(\mathbf{a} \cdot \mathbf{a}^{\star}\right) \mathbf{a}^{\star}$ and $\overline{\mathbf{b}}=\mathbf{b}-\left(\mathbf{b} \cdot \mathbf{a}^{\star}\right) \mathbf{a}^{\star}$, with
$\mathbf{a}=\mathbf{Q a} \mathbf{a}_{0}, \quad \mathbf{a}^{\star}=\mathbf{Q a} \mathbf{a}_{0}^{\star}$.
The sign of the sine of the transmission angle is given by

$$
\begin{equation*}
\operatorname{sgn}(\mu)=\operatorname{sgn}\left[(\overline{\mathbf{a}} \times \overline{\mathbf{b}}) \cdot \mathbf{a}^{\star}\right] \tag{18}
\end{equation*}
$$

## Example 1

Prescribed poses described by axes and angles of rotations with reference to the initial pose.

Table: Five poses for Example 1

| $\phi_{j}[\mathrm{rad}]$ | $\mathbf{e}^{T}$ | $\left[\beta_{1}, \beta_{2}, \beta_{3}\right][\mathrm{deg}]$ |
| :---: | :---: | :---: |
| 0 | $[0,0,1]$ | $[0,0,0]$ |
| 0.2034 | $[-0.0449,-0.5133,-0.8569]$ | $[-10.0,-6.0,0]$ |
| 1.1957 | $[0.1827,0.7709,-0.6101]$ | $[-51.0,-52.0,-12.0]$ |
| 1.1932 | $[0.5212,0.8414,-0.1422]$ | $[15.0,-56.0,47.0]$ |
| 1.0512 | $[0.5384,0.8114,0.2271]$ | $[33.0,-40.0,47.0]$ |

Note: $\beta_{1}, \beta_{2}$ and $\beta_{3}$ are the corresponding angles of longitude and latitude, respectively, which are also given with the purpose of helping the visualization of the prescribed attitudes.

## Example 1 (cont'd)

Four sets of real solutions are found from the contour plots of angles of longitude and latitude

(a)

(b)

|  | $\mathbf{a}_{0}\left(\right.$ or $\left.\mathbf{a}_{0}^{\star}\right)$ | $\mathbf{b}\left(\right.$ or $\left.\mathbf{b}^{\star}\right)$ |
| :---: | :---: | :---: |
| $\# 1$ | $[0.7085,-.6418,-.2932]$ | $[0.2640,-.6636,-.6998]$ |
| $\# 2$ | $[0.0385,0.3163,0.9478]$ | $[0.1143,0.7263,-.6777]$ |
| $\# 3$ | $[0.1642,0.6977,0.6972]$ | $[0.5218,0.8413,-.1403]$ |
| $\# 4$ | $[0.8077,0.1493,0.5702]$ | $[0.9524,-.2535,0.1686]$ |

## Six spherical linkages for Example 1

Altogether, six linkages can be synthesized for this example.


Note:

- All task orientations shown in yellow
- The frames in red show the orientations of the coupler
- Branch defects were detected for M3 and M4.


## Example 2: Synthesis with one P-joint

Five poses and solutions to the linkage with one P-joint

| $\phi_{j}[\mathrm{rad}]$ | $\mathbf{e}^{T}$ | $\left[\beta_{1}, \beta_{2}, \beta_{3}\right][\mathrm{deg}]$ |
| :---: | :---: | :---: |
| 0 | $[0,0,1]$ | $[0,0,0]$ |
| 0.2563 | $[-0.2280,-0.4553,-0.8606]$ | $[-12.5,7.0,-2.6]$ |
| 1.1307 | $[-0.0578,0.2370,-0.9697]$ | $[-64.2,-10.5,-10.8]$ |
| 1.1938 | $[0.5049,0.8505,-0.1468]$ | $[14.3,-56.9,45.6]$ |
| 1.3665 | $[0.7119,0.6601,0.2393]$ | $[45.1,-30.7,73.2]$ |


|  | $\mathbf{a}_{0}$ | $\mathbf{b}$ |
| :---: | :---: | :---: |
| $\# 1$ | $[0.2845,0.3863,0.8773]$ | $[0.1219,-.7089,-.6946]$ |
| $\# 2$ | $[0.7226,0.5295,0.4442]$ | $[0.2309,0.4566,0.8591]$ |
| $\# 3$ | $[0.9573,-.2433,0.1555]$ | $[0.8134,0.1643,0.5579]$ |
|  | $\mathbf{a}_{0}^{\star}$ | $\mathbf{b}^{\star}$ |
| $\# 4$ | $[0.5221,0.8442,-.1208]$ | $[0.0655,0.1015,0.9926]$ |

## Mechanism synthesized for Example 2



Figure: One synthesized mechanism with a P-joint, showing three link attitudes

## Conclusions

- A robust solution of synthesis equation for the spherical Burmester problem was formulated
- A semigraphical solution method was developed
- Branch defect detection and synthesis with one P-joint were considered
- The method was demonstrated with two examples
- Future work? spatial Burmester problem


## References

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